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Quantifying Demand Balancing in Bidirectional Low Temperature Networks

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Abstract

Bidirectional low temperature networks (BLTNs) are a promising technology to drive decarbonization of the heating and cooling sector. The energy efficiency of a BLTN strongly depends on the heating and cooling demands of the connected buildings and their simultaneity. In order to evaluate the efficiency of a BLTN, a novel metric, called *Demand Overlap Coefficient* (DOC), is presented in this paper. The DOC can be calculated for individual buildings or a district and describes which proportion of heating and cooling demands can be balanced *inside* buildings and *between* buildings. The DOC is calculated with time series for heating and cooling demands, taking special account of their simultaneity. For a real-world use case in Germany, it is shown that 25 % of heating and cooling demands can be balanced in buildings and 45 % can be balanced by the BLTN between buildings. The DOC is evaluated for 63 demand scenarios. Correlations show that for districts with a DOC larger than 0.45, BLTNs have lower annualized costs than a state-of-the-art heating and cooling system. BLTNs have higher exergy efficiencies for districts with

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DOCs larger than 0.3. With the derived correlations, the DOC serves as a practical key metric in the planning process of BLTNs in order to identify and assemble building clusters with complementary demand profiles.

Keywords:

Bidirectional low temperature network, 5GDHC, Waste heat, District heating, District cooling, Demand balancing

1. Introduction

Waste heat recovery is one key approach to increase the efficiency of energy systems, and thus to reduce fossil fuel consumption and carbon emissions [1]. The higher the temperature level of waste heat, the larger the economic and energy saving potential. For industrial processes with high process temperatures, waste heat recovery is a key approach to reduce costs and increase process efficiencies [2]: Recovered heat can be used in the process itself to increase its efficiency [3] or reused by other industrial or commercial processes [5], e.g. organic rankine cycle for electricity generation.

In the heating and cooling sector of commercial and residential buildings, waste heat, e.g. from cooling applications, usually remains unused due to its low temperature level. For example in data centers, low-grade waste heat is often dissipated ([6], [7]) instead of reusing it for other purposes, like district heating ([8], [9]).

Nevertheless, the energy saving potential of recovering low-grade waste heat is enormous since in urban districts the amount of waste heat can reach 50 – 120 % of the annual heat demand [10]. In addition, the importance of using low-grade waste heat sources will increase in future, since space cooling

19 demands, as one major waste heat source, is expected to rise substantially in
20 the coming decades [11]. Nowadays, space cooling is often provided by indi-
21 vidual air conditioning units which dissipate waste heat to the environment.

22 To unlock urban waste heat potentials, a thermal connection between
23 buildings through a central distribution infrastructure is crucial ([12], [13])
24 and recently gains more interest ([14], [15]). In the field of low temperature
25 distribution grids, one promising concept are bidirectional low temperature
26 networks (BLTNs) ([16], [17]), which are also referred to as *5th Generation*
27 *District Heating and Cooling* (5GDHC) networks ([18], [19], [20], [21]), *cold*
28 *district heating* ([22]), *low-temperature district heating and cooling networks*
29 ([23], [24]) or *balanced energy networks* [25]. These networks are the latest
30 development stage of district heating and follow the trend towards lower
31 operating temperatures, as illustrated in Fig. 1.

32 BLTNs do not have a supply and return pipe, but a warm and cold pipe
33 with a temperature difference of about 5 – 10 K. Both pipes are operated
34 at temperatures close to the surrounding (5 – 35 °C). Due to the low tem-
35 peratures, heating demands of buildings cannot be covered directly, and a
36 heat pump is installed in each building to raise the temperature to the level
37 required by the building’s heating system. In heating mode, a heat pump
38 takes water from the warm pipe and uses it as heat source. The cooled down
39 water from the evaporator of the heat pump is then discharged to the cold
40 pipe. If the building has a cooling demand, the fluid flow is opposite: Water
41 from the cold pipe is warmed up in the building and discharged to the warm
42 pipe. A detailed description is provided in [18] and [20]. One key advantage
43 of BLTNs is that waste heat from a building with cooling demand can be

44 reused to cover heating demands of other buildings. Thus, heating and cool-
 45 ing demands of a district are balanced out to some extent. Blacha et al. [26]
 46 and Rogers et al. [27] show that the amount of recovered heat, i.e. the share
 47 of balanced demands, has a strong impact on the performance of BLTNs.
 48 Boesten et al. suggest to consider demand profiles of buildings in urban
 49 planning in order to create neighborhoods with complementary heating and
 50 cooling demand profiles [20].

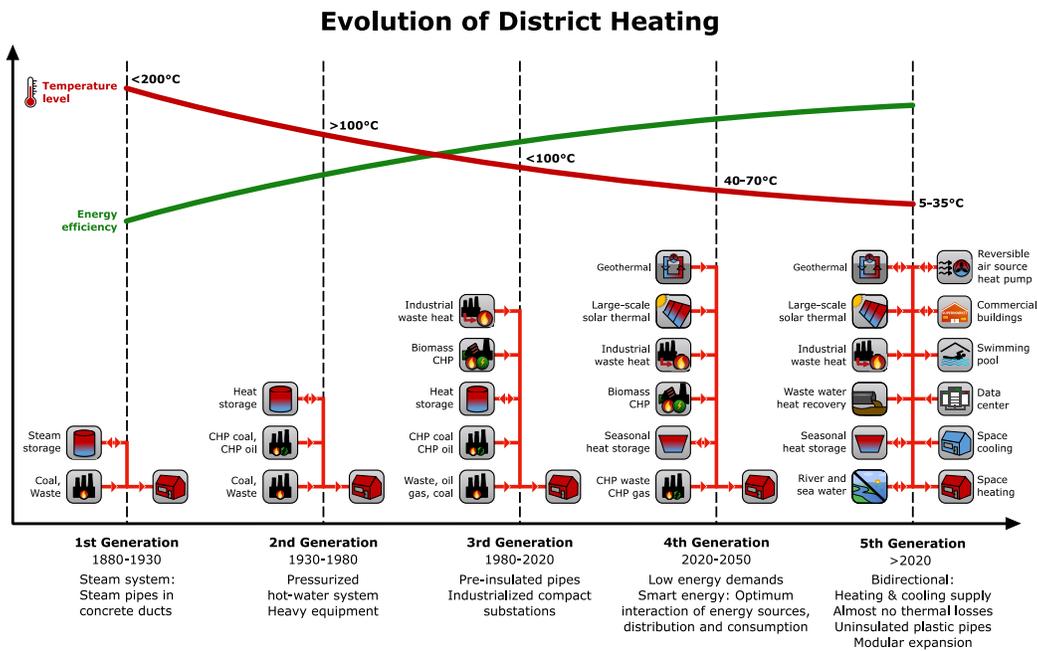


Figure 1: 5th Generation District Heating and Cooling networks are the latest stage in the evolution of district heating. Illustration based on Lund et al. [28].

51 *1.1. Key metrics for quantifying waste heat potential*

52 In this section, an overview about recently presented key metrics to quan-
 53 tify waste heat potential is provided: For data centers, Wahlroos et al. ([29],

54 [30]) present the *Energy Reuse Factor*, which describes the ratio of reused
55 energy to total energy consumption for data centers.

56 Papapetrou et al. [31] introduce the *Waste Heat Fraction*. It is the ratio
57 of waste heat potential to total heat consumption for a specific industrial
58 sector and temperature band. However, their approach focuses on industrial
59 processes for which waste heat sources and heating demands are usually not
60 shifted by seasonal effects.

61 Fang et al. [5] discuss the potential of low temperature district heat-
62 ing networks for recovering industrial waste heat in northern China. For
63 estimating the waste heat potential, they introduce the *Coefficient of Poten-*
64 *tial*, which describes the ratio of “theoretical amount of waste heat from all
65 intensive sectors in district heating regions to district heating energy con-
66 sumption”. However, they do not elaborate this approach further regarding
67 temporal availability or temperature levels of heat sources and demands.

68 Persson et al. [32] present key metrics to quantify heat recovery. They in-
69 troduce the *Heat Recovery Rate* and *Heat Utilization Rate*. The heat recovery
70 rate measures the proportion of recovered waste heat. The heat utilization
71 rate indicates the extent to which recovered excess heat is actually used to
72 cover heat demands.

73 Woolley et al. [33] consider the temporal distribution of waste heat in
74 relation to the heat demand. For this purpose, they define an overlap function
75 which describes the minimum of two exergy flows: The exergy flow of the
76 (waste) heat source and the exergy flow of the heat sink. By integrating the
77 overlap function over time, they calculate the total amount of exergy that
78 can be recovered from the waste heat. Furthermore, they define a *Recovery*

79 *Index* as the ratio of recoverable exergy to total available exergy.

80 For BLTNs, no metric has yet been developed which quantifies the waste
81 heat potential in these systems. However, Pass et al. [34] present the *Di-*
82 *versity index* to quantify the diversity of heating and cooling demands: A
83 large Diversity index indicates that heating and cooling demands are about
84 the same magnitude for a given point in time, which suggests that a large
85 proportion of waste heat can be recovered. A Diversity index of zero indi-
86 cates that only heating or only cooling demands occur at a given point in
87 time and no waste heat can be recovered. The numeric value of the Diversity
88 index has no physical meaning. In particular, no conclusions can be drawn
89 about the proportion of balanced demands. Pass et al. consider exergy effi-
90 ciency in order to evaluate the performance of a district energy system with
91 BLTN. However, no conclusions about the profitability of the system, e.g.
92 by considering total annualized costs, are drawn.

93 1.2. Contributions

94 In this paper, a novel metric is presented that quantifies the balancing
95 potential of heating and cooling demands in districts with BLTN and at the
96 same time characterizes the demand structure of a district. The metric is
97 called *Demand Overlap Coefficient* (DOC), as it is a measure of the overlap
98 of heating and cooling demand profiles. In contrast to the metrics presented
99 in [5], [29], [30], [31] and [32], the DOC takes into account the simultaneity of
100 heat sources and demands. Unlike the Diversity index presented in [34], the
101 DOC has a physical meaning and can be interpreted intuitively: It describes
102 the proportion of thermal demands that can potentially be balanced in a
103 district energy system with BLTN. The DOC can be determined for different

104 subsystems and indicates the balancing potential *inside* buildings or in the
105 BLTN *between* buildings.

106 Based on 63 demand scenarios, correlations between the DOC and key
107 performance indicators, like exergy efficiency and total annualized costs, are
108 derived. The evaluation of the system performance is conducted with a linear
109 program adapted from [35]. It is shown that the DOC can serve as a practical
110 key metric to identify clusters of buildings with complementary heating and
111 cooling demands and to estimate the profitability and efficiency of a BLTN
112 in the early planning phase.

113 *1.3. Paper organization*

114 The structure of this paper is as follows: In Section 2.1, the DOC with
115 all related equations is introduced. The methodology to evaluate the perfor-
116 mance of a district energy system with BLTN is presented in Section 2.2. In
117 Section 3, the DOC is determined for a real-world use case and the balancing
118 of demands is explained in detail. In Section 4, 63 different demand sce-
119 narios are analyzed and correlations between the DOC and key performance
120 indicators are derived. Finally, conclusions are provided in Section 5.

121 **2. Methodology**

122 In the following Section 2.1, the definitions and equations of the DOC are
123 derived. The methodology for evaluating the performance of district energy
124 systems with BLTNs is explained in Section 2.2.

125 *2.1. Demand Overlap Coefficient*

126 In urban districts, low temperature waste heat often results from cooling
127 applications (e.g. space cooling with compression chillers). This low-grade
128 waste heat can be directly fed into a BLTN to cover heating demands of other
129 buildings. This way, heating and cooling demands are balanced out to some
130 extent. The proportion of heating demands that can be covered by waste heat
131 sources depends on the amount and temperature level of the waste heat but
132 also on the simultaneity of demands and sources. In this section, the DOC is
133 introduced to evaluate the waste heat potential for districts with BLTN. More
134 precisely, the DOC describes the proportion of heating and cooling demands
135 that can be canceled out in the system. In the following Section 2.1.1, the
136 *District DOC* is introduced which characterizes the demand structure of a
137 district. In the subsequent Sections 2.1.2 and 2.1.3, two further DOCs for
138 demand balancing *inside* and *between* buildings are derived. Section 2.1.4
139 presents the relationships between all three DOCs.

140 *2.1.1. District DOC*

141 The District DOC is a metric to quantify the temporal correspondence
142 of heating and cooling demands in a district, i.e. to what extent heating
143 and cooling demands match in time and magnitude. The District DOC is
144 calculated solely on the basis of building energy demands and no knowledge
145 about the energy system is required. Fig. 2 shows exemplary heating and
146 cooling demand profiles ($\dot{Q}_{h,dem}(t) / \dot{Q}_{c,dem}(t)$). The balancing heat flow
147 (that results from the cooling process and covers a proportion of the heating
148 demand) is

$$\dot{Q}_{bal}(t) = \min \left\{ \dot{Q}_{h,dem}(t), \dot{Q}_{c,dem}(t) \right\} \quad (1)$$

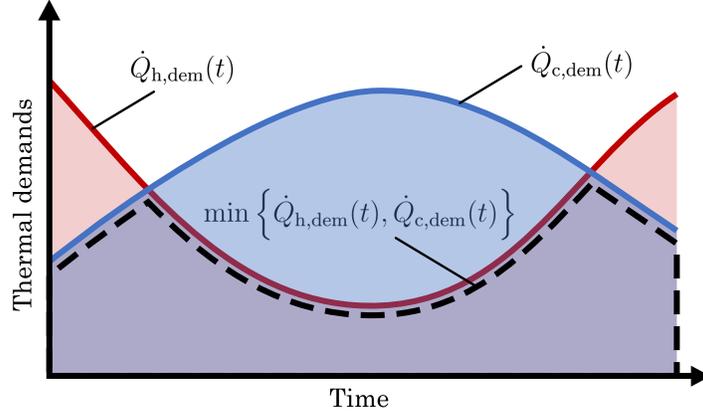


Figure 2: Heating (red) and cooling demands (blue) are depicted over time. The Demand Overlap Coefficient quantifies the overlap of both demands expressed by $\min \{ \dot{Q}_{h,dem}(t), \dot{Q}_{c,dem}(t) \}$ (black dashed line).

149 and is indicated by the dashed black line in Fig. 2. The proportion of the
 150 balanced heating and cooling demands for one point in time t is

$$\frac{2 \cdot \min \{ \dot{Q}_{h,dem}(t), \dot{Q}_{c,dem}(t) \}}{\dot{Q}_{h,dem}(t) + \dot{Q}_{c,dem}(t)} \quad (2)$$

151 Here, the factor 2 takes into account that one unit of balanced demands
 152 equals one unit of heating plus one unit of cooling demand, and thus ensures
 153 that the proportion ranges between 0 and 1. For an exemplary heating
 154 demand of $\dot{Q}_{h,dem} = 6$ kW and a cooling demand of $\dot{Q}_{c,dem} = 2$ kW, this
 155 proportion is

$$\Phi = \frac{4}{6 + 2} = \frac{1}{2} \quad (3)$$

156 The proportion of the total balanced demands during a time interval is

$$\Phi = \frac{\int 2 \cdot \min \{ \dot{Q}_{h,dem}(t), \dot{Q}_{c,dem}(t) \} dt}{\int \dot{Q}_{h,dem}(t) + \dot{Q}_{c,dem}(t) dt} \quad (4)$$

157 and is therefore two times the area under the dashed line divided by the sum
 158 of the areas under the red and blue line. For discrete, equally spaced time
 159 intervals $t \in T$ follows:

$$\Phi = \frac{2 \cdot \sum_{t \in T} \min \{ \dot{Q}_{h,dem,t}, \dot{Q}_{c,dem,t} \}}{\sum_{t \in T} (\dot{Q}_{h,dem,t} + \dot{Q}_{c,dem,t})} \quad (5)$$

160 By summing up all heating and cooling demands of all buildings $b \in B$ in a
 161 district, the definition of the District DOC is obtained:

$$\Phi_{\text{distr}} = \frac{2 \cdot \sum_{t \in T} \min \left\{ \sum_{b \in B} \dot{Q}_{h,dem,b,t}, \sum_{b \in B} \dot{Q}_{c,dem,b,t} \right\}}{\sum_{t \in T} \sum_{b \in B} (\dot{Q}_{h,dem,b,t} + \dot{Q}_{c,dem,b,t})} \quad (6)$$

162 The DOC ranges between 0 and 1. A DOC of 0 means that heating and
 163 cooling demand profiles do not overlap at all, a DOC of 1 means they match
 164 exactly. However, the District DOC only estimates the demand balancing
 165 potential due to the fact that the temperature level of waste heat from chillers
 166 is usually not high enough to cover heating demands directly. As a result, in
 167 districts with BLTN, heat pumps and chillers are installed to raise or lower
 168 the temperature of waste heat flows. Therefore, in the following Section 2.1.2,
 169 this effect is taken into account when the DOC for a building energy system
 170 is derived.

171 2.1.2. Building energy system DOC (BES DOC)

172 The building energy system DOC (BES DOC) describes the balancing
 173 potential for a building energy system (BES) in which heating and cooling
 174 demands overlap. In the following, a BES with a heat pump and a chiller is
 175 considered. However, the definition of the BES DOC can be applied to any

176 BES configuration which use heat and cold from a thermal network. (For
 177 other BES configurations, Eqs. (7) and (8) have to be adapted to express the
 178 net heating and cooling demand of the BES). In Fig. 3, the considered BES
 179 with heat pump and chiller as well as the heating and cooling demands of
 180 the buildings ($\dot{Q}_{h,dem,b,t}$ and $\dot{Q}_{c,dem,b,t}$) are illustrated. The heat flow to the
 181 evaporator of the heat pump (or more general the heat demand of the BES)
 182 is denoted by $\dot{Q}_{h,BES,b,t}$ and waste heat from the compression chiller (cooling
 183 demand of the BES) by $\dot{Q}_{c,BES,b,t}$. These heat flows can be expressed with
 184 their respective COPs:

$$\dot{Q}_{h,BES,b,t} = \dot{Q}_{h,dem,b,t} \left(1 - \frac{1}{COP_{HP}} \right) \quad (7)$$

185

$$\dot{Q}_{c,BES,b,t} = \dot{Q}_{c,dem,b,t} \left(1 + \frac{1}{COP_{CC}} \right) \quad (8)$$

186 Waste heat from the chiller is used as heat source for the heat pump. This
 187 balancing heat flow is

$$\dot{Q}_{BES,bal,b,t} = \min \left\{ \dot{Q}_{h,BES,b,t}, \dot{Q}_{c,BES,b,t} \right\} \quad (9)$$

188 As a result, the BES DOC for the a building b is

$$\Phi_{BES,b} = \frac{2 \cdot \sum_{t \in T} \min \left\{ \dot{Q}_{h,BES,b,t}, \dot{Q}_{c,BES,b,t} \right\}}{\sum_{t \in T} \left(\dot{Q}_{h,BES,b,t} + \dot{Q}_{c,BES,b,t} \right)} \quad \forall b \in B \quad (10)$$

189 and describes proportion of balanced heating and cooling demands in a build-
 190 ing. The BES DOCs of a set of buildings can be averaged and expressed by
 191 a mean BES DOC:

$$\bar{\Phi}_{BES} = \frac{2 \cdot \sum_{b \in B} \sum_{t \in T} \min \left\{ \dot{Q}_{h,BES,b,t}, \dot{Q}_{c,BES,b,t} \right\}}{\sum_{b \in B} \sum_{t \in T} \left(\dot{Q}_{h,BES,b,t} + \dot{Q}_{c,BES,b,t} \right)} \quad (11)$$

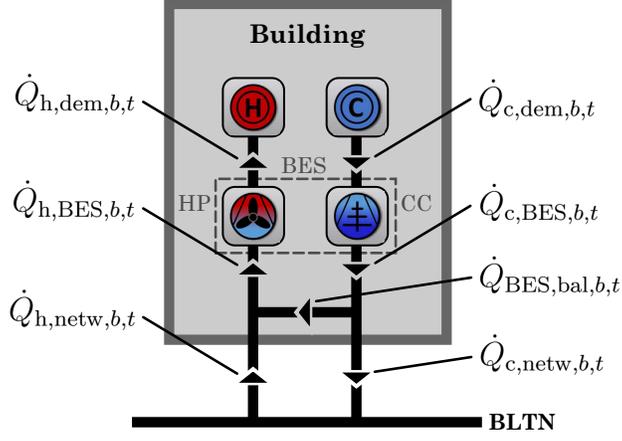


Figure 3: Demand balancing in buildings: Waste heat from the compression chiller (CC) is used as heat source for the heat pump (HP). Residual loads are taken from the thermal network ($\dot{Q}_{h,netw,b,t}$ and $\dot{Q}_{c,netw,b,t}$).

192 Equivalently, the mean BES DOC ($\bar{\Phi}_{BES}$) can be expressed with the BES
 193 DOCs of the individual buildings ($\Phi_{BES,b}$):

$$\bar{\Phi}_{BES} = \frac{\sum_{b \in B} \left(\Phi_{BES,b} \sum_{t \in T} \left(\dot{Q}_{h,BES,b,t} + \dot{Q}_{c,BES,b,t} \right) \right)}{\sum_{b \in B} \sum_{t \in T} \left(\dot{Q}_{h,BES,b,t} + \dot{Q}_{c,BES,b,t} \right)} \quad (12)$$

194 2.1.3. Network DOC

195 The Network DOC is a measure for the overlap of the heat flows from the
 196 network to the buildings and vice versa and therefore describes the balancing
 197 of heating and cooling demands between buildings. In Fig. 3, the heat flow
 198 from the BLTN to the building is

$$\dot{Q}_{h,netw,b,t} = \dot{Q}_{h,BES,b,t} - \dot{Q}_{BES,bal,b,t} \quad (13)$$

199 and from the building to the BLTN is

$$\dot{Q}_{c,netw,b,t} = \dot{Q}_{c,BES,b,t} - \dot{Q}_{BES,bal,b,t} \quad (14)$$

200 For the Network DOC, these net demands of all buildings (heating demand:
 201 $\sum_{b \in B} \dot{Q}_{h,netw,b,t}$; cooling demand: $\sum_{b \in B} \dot{Q}_{c,netw,b,t}$) are considered. The Net-
 202 work DOC is then defined as

$$\Phi_{netw} = \frac{2 \cdot \sum_{t \in T} \min \left\{ \sum_{b \in B} \dot{Q}_{h,netw,b,t}, \sum_{b \in B} \dot{Q}_{c,netw,b,t} \right\}}{\sum_{t \in T} \left(\sum_{b \in B} \dot{Q}_{h,netw,b,t} + \sum_{b \in B} \dot{Q}_{c,netw,b,t} \right)} \quad (15)$$

203 Demands, which cannot be balanced in the BLTN, are covered by external
 204 supply systems, e.g. an energy hub (EH). In the following, these residual
 205 demands of the network are denoted by $\dot{Q}_{h,EH,t}$ and $\dot{Q}_{c,EH,t}$.

206 2.1.4. Relationship between different DOCs

207 In this section, the DOCs are summarized and it is shown how they are
 208 related to each other. The definitions of the District DOC, BES DOC and
 209 Network DOC are illustrated in Fig. 4. The BES DOC and Network DOC
 210 couple the thermal demands of the different subsystems with each other: The
 211 net heating and cooling demands of building b are coupled with the thermal
 212 demands of the BES by the BES DOC:

$$\sum_{t \in T} (\dot{Q}_{h,netw,b,t} + \dot{Q}_{c,netw,b,t}) = (1 - \Phi_{BES,b}) \sum_{t \in T} (\dot{Q}_{h,BES,b,t} + \dot{Q}_{c,BES,b,t}) \quad (16)$$

213 Likewise, the net building demands and the residual network demands are
 214 coupled by the Network DOC:

$$\sum_{t \in T} (\dot{Q}_{h,EH,t} + \dot{Q}_{c,EH,t}) = (1 - \Phi_{netw}) \sum_{b \in B} \sum_{t \in T} (\dot{Q}_{h,netw,b,t} + \dot{Q}_{c,netw,b,t}) \quad (17)$$

215 By inserting Eq. (16) in Eq. (17), an expression is obtained that couples the
 216 thermal demands of the BESs with the residual network demands:

$$\sum_{t \in T} (\dot{Q}_{h,EH,t} + \dot{Q}_{c,EH,t}) = (1 - \Phi_{netw})(1 - \bar{\Phi}_{BES}) \sum_{b \in B} \sum_{t \in T} (\dot{Q}_{h,BES,b,t} + \dot{Q}_{c,BES,b,t}) \quad (18)$$

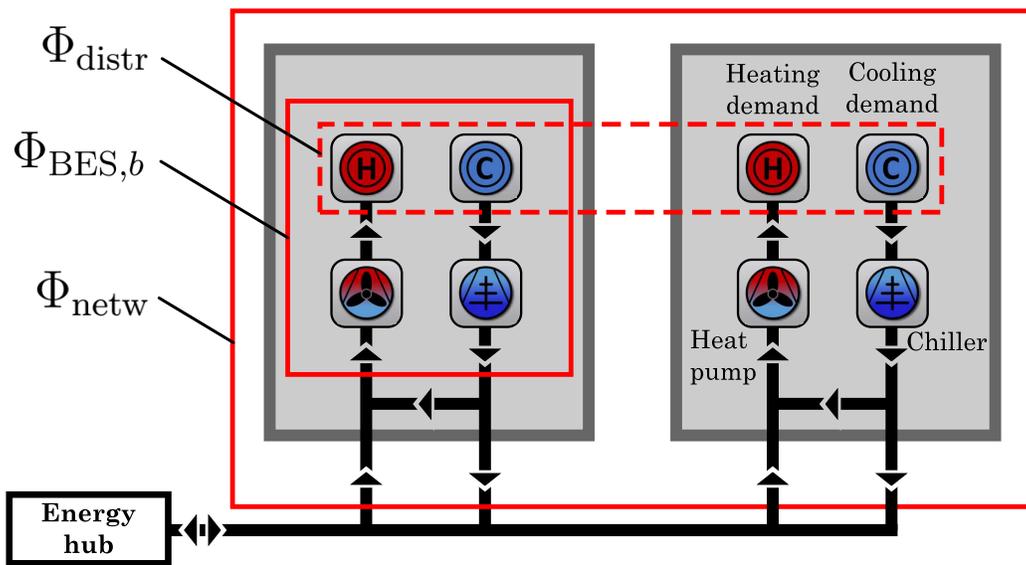


Figure 4: The simultaneity of heating and cooling demands can be calculated for three subsystems: Cumulated demands of all connected buildings (Φ_{distr}), demands of an individual building energy system ($\Phi_{\text{BES},b}$) and cumulated net demand of all connected buildings (Φ_{netw}).

217 In Section 4, it is shown that a correlation between the Network DOC, mean
218 BES DOC and District DOC exists, from which the approximation

$$(1 - \Phi_{\text{netw}})(1 - \bar{\Phi}_{\text{BES}}) \approx (1 - \Phi_{\text{distr}}) \quad (19)$$

219 is derived.

220 2.1.5. *Limitations*

221 The definition of the DOC is subject to simplifications: A limitation of the
222 Network DOC is that the spatial distribution of heat sources and sinks is not
223 taken into account, and therefore hydraulic network limitations are neglected.
224 As a result, an ideal exchange of waste heat among buildings is assumed. This
225 simplification is considered justified for small districts. However, for large
226 thermal networks with a large number of connected buildings, the effect of
227 hydraulic limitations becomes more relevant. Moreover, by assuming an ideal
228 exchange of waste heat, heat losses to the ground are neglected.

229 The Network DOC is particularly meaningful for systems with no or only
230 small storage capacities. For systems with large thermal storages, like sea-
231 sonal storages, the simultaneity of demands is of less relevance since storages
232 enable demand balancing across different time periods. If thermal storages
233 are used in the network, the temporal resolution of the periods for calculating
234 the Network DOC should be adjusted to the storage capacity. For example,
235 if thermal storages balance day-night cycles, the calculation of the Network
236 DOC should be based on time periods with a length of one day. Similarly, the
237 thermal inertia of the water mass in the network can function as a thermal
238 storage. However, this effect can be neglected for most of the BLTNs since
239 the water volume of the network is small, especially compared to the water

240 volume of a central storage.

241 *2.1.6. Application in planning of BLTNs*

242 The DOC is a metric to support the development of sustainable urban dis-
243 tricts, and especially the planning of district energy systems with BLTNs. In
244 an early design phase of districts, urban planners can use the DOC to ensure
245 that the waste heat potential of a district can be exploited to the greatest
246 extend possible. In this context, the DOC can help to answer questions like:

- 247 • Which building types are considered for a district and what is a benefi-
248 cial composition from an energy point of view? Only dwellings or also
249 commercial buildings? If only dwellings are planned, a BLTN tends
250 to be less profitable since waste heat and heat demands do not occur
251 simultaneously.
- 252 • How are different buildings grouped in a district, e.g. mixed-use zones
253 or homogeneous structure? Mixed-use zones lead to smaller clusters
254 with complementary demands which can be interconnected with small
255 networks.

256 In the technical planning process of a BLTN, the DOC is a practical
257 metric to answer questions like:

- 258 • Should a more distant building connected to the network or should it
259 be supplied by an individual energy system?
- 260 • Is it beneficial to connect buildings outside of the district to the BLTN?
261 For example, connecting a nearby supermarket or factory offering waste
262 heat in winter can be beneficial for the overall system performance.

263 • Should different BLTNs in a city be connected or stay separated? To
264 answer this question, the DOC can be calculated with the cumulated
265 demands of both BLTN clusters. A connection of two BLTNs can be
266 beneficial if the DOC of the connected clusters is larger than the DOC
267 of the single clusters.

268 *2.2. Evaluation of system performance*

269 In Section 4, correlations between the DOCs and the performance of dis-
270 trict energy systems with BLTN are investigated. In order to determine
271 the system performance for many different use cases, an optimization model
272 (linear programming) is used. The model is briefly explained in the follow-
273 ing Section 2.2.1. In Section 2.2.2, the key performance indicators for the
274 evaluations in this study are introduced.

275 *2.2.1. Linear program for system calculation*

276 The linear program used in this study is adapted from a model presented
277 by Wirtz et al. [35]. The linear program determines the optimal selection
278 and sizing of all system components as well as their optimal operation by
279 minimizing total annualized costs. The optimization model is described by
280 two superstructures: one for the energy hub and one for the building energy
281 systems. From the technologies of the superstructure, the optimal system
282 configuration is selected in the optimization.

283 The superstructure of the energy hub is depicted in Fig. 5. It contains a
284 reversible air source heat pump which heats or cools the BLTN. The COP of
285 the heat pump is calculated a priori based on the ambient air temperature.
286 As storage units, an ice thermal energy storage (ITES) and a battery (BAT)

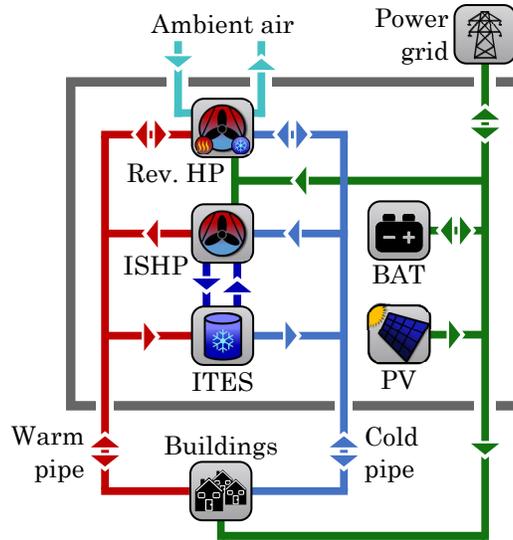


Figure 5: Optimization superstructure of the energy hub: Heat is generated by a reversible air source heat pump (Rev. HP) and the ice storage heat pump (ISHP). The ice thermal energy storage (ITES) and reversible heat pump can cool the thermal network. Photovoltaic modules (PV) generate electricity. A battery (BAT) can be installed to increase the self consumption rate.

287 are considered. Surplus heat from the BLTN can be used to regenerate the
 288 ice storage. A heat pump connected to the ice storage (ISHP) can extract
 289 heat from the storage and heat the BLTN. For the water in the ice storage, a
 290 constant temperature of 0°C is assumed. Furthermore, photovoltaic modules
 291 (PV) are part of the superstructure. The superstructure of the building
 292 energy systems is depicted in Fig. 6. It contains a heat pump, electric boiler,
 293 heat storage, chiller and a heat exchanger for direct cooling. The objective
 294 function and all constraints of the linear program are listed in Appendix B.

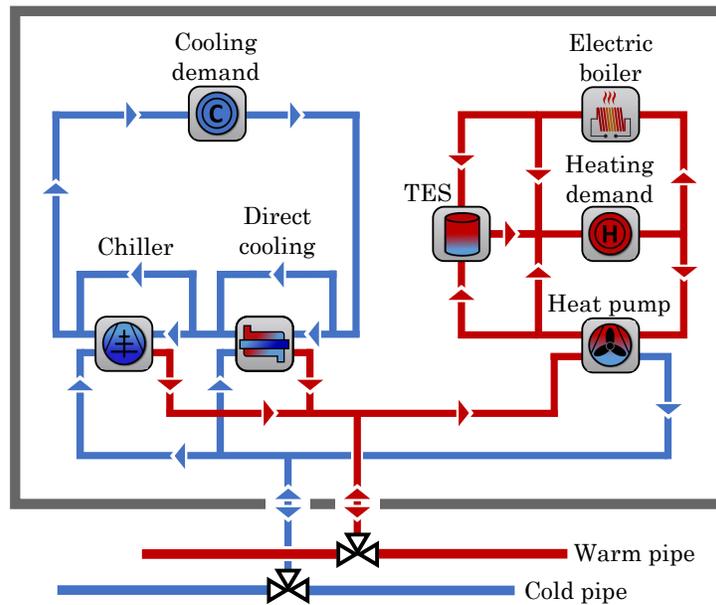


Figure 6: Optimization superstructure of building energy systems (based on [35]): Heating demands are covered by a water-to-water heat pump, electric boiler and thermal energy storage (TES). Cooling demands are covered by a compression chiller and a heat exchanger for direct cooling with the cold pipe of the BLTN.

295 *2.2.2. Key performance indicators*

296 Based on the system operation (determined with the linear program),
297 four key performance indicators (KPIs) are considered in this study which
298 are introduced in the following.

299

300 **Specific supply costs for heating and cooling**

301 The objective function of the optimization model are total annualized costs
302 (*TAC*). The specific costs for the heating and cooling supply are

$$c_{\text{tot}} = \frac{TAC}{Q_{\text{h,dem}}^{\text{tot}} + Q_{\text{c,dem}}^{\text{tot}}} \quad \text{in} \quad \frac{\text{EUR}}{\text{MWh}} \quad (20)$$

303 in which $Q_{\text{h,dem}}^{\text{tot}}/Q_{\text{c,dem}}^{\text{tot}}$ are the annual heating and cooling demands.

304

305 **Exergy efficiency**

306 Following the exergetic assessment of districts in [36] and [37], the exergy
307 efficiency is used as another KPI. The exergy efficiency of a system is the
308 ratio of total useful exergy to the total exergy expenditures:

$$\eta_{\text{ex}} = \frac{E_{\text{h,dem}}^{\text{tot}} + E_{\text{c,dem}}^{\text{tot}} + W_{\text{feed-in}}^{\text{tot}}}{W_{\text{grid}}^{\text{tot}} + W_{\text{PV}}^{\text{tot}}} \quad (21)$$

309 Here, $E_{\text{h,dem}}^{\text{tot}}/E_{\text{c,dem}}^{\text{tot}}$ denote the exergy of the annual heating and cooling
310 demands of all buildings. The total electricity fed into the electricity grid
311 ($W_{\text{feed-in}}^{\text{tot}}$) is considered useful exergy, the total electricity imported from the
312 grid ($W_{\text{grid}}^{\text{tot}}$) is considered expenditure. The electricity generated by photo-
313 voltaics is considered expenditure ($W_{\text{PV}}^{\text{tot}}$). Based on [38], the exergy of heat

314 and cold flows is:

$$E_{h,dem}^{tot} = Q_{h,dem}^{tot} \left(1 - \frac{T_{ref}}{T_{h,sup}} \right) \quad (22)$$

$$E_{c,dem}^{tot} = Q_{c,dem}^{tot} \left(\frac{T_{ref}}{T_{c,sup}} - 1 \right) \quad (23)$$

315 Here, the $T_{h,sup}/T_{c,sup}$ denote the supply temperature of the heating/cooling
 316 system in the building and T_{ref} denotes the reference temperature (in this
 317 study: $T_{ref} = 25^\circ\text{C} = 298.15\text{ K}$).

318

319 **System COP**

320 The entire energy supply system with the reversible heat pump in the energy
 321 hub and the heat pumps in the buildings can be considered as one cascaded
 322 heat pump process. Therefore, as a third KPI, an overall coefficient of per-
 323 formance of the energy supply system (*System COP*) is considered. The
 324 definition is based on the *Figure of merit* introduced by Rosen et al. [39]:

$$COP_{Sys} = \frac{Q_{h,dem}^{tot} + Q_{c,dem}^{tot} + W_{feed-in}^{tot}}{W_{grid}^{tot} + W_{PV}^{tot}} \quad (24)$$

325 Here, $Q_{h,dem}^{tot}$ and $Q_{c,dem}^{tot}$ denote the total heating and cooling demands,
 326 $W_{feed-in}^{tot}$ the total electricity fed into the electricity grid, W_{grid}^{tot} the total elec-
 327 tricity imported from the grid and W_{PV}^{tot} the total PV generation.

328

329 **Specific CO₂ emissions**

330 The specific CO₂ emissions of the district energy system are

$$e_{tot} = \frac{W_{grid}^{tot} e_{grid}}{Q_{h,dem}^{tot} + Q_{c,dem}^{tot}} \quad \text{in} \quad \frac{\text{tCO}_2}{\text{MWh}} \quad (25)$$

331 Here, e_{grid} is the CO₂ factor of the electricity grid mix.

332 *2.2.3. Reference system*

333 In order to evaluate the performance of the BLTN system, it is compared
334 with a reference system which represents a state-of-the-art solution. For heat-
335 ing and cooling supply in the reference system, each building is equipped with
336 an air source heat pump, electric boiler, heat storage as well as a compression
337 chiller. Like in the BLTN system, PV modules are installed. The reference
338 system does not have an energy hub or thermal network, and therefore heat
339 exchange between buildings is not possible.

340 **3. Use case**

341 In this section, the presented methodology is applied to a real-world use
342 case. Section 3.1 provides a brief description of the use case. In Section 3.2,
343 the performance of the system is evaluated according to four KPIs and com-
344 pared to a reference system. In Section 3.3, the three DOCs are determined
345 for the use case and the effect of demand balancing is described in detail.

346 *3.1. Use case description*

347 The use case (adopted from [35]) is a reasearch campus in Germany, for
348 which 17 buildings are considered. A monitoring system has been installed
349 on the campus, which logs heating and cooling demands at substations in
350 all buildings with a sub-hourly resolution. For this study, raw data for one
351 year has been aggregated to hourly demand time series (8760 data points
352 for each demand profile). An overview of heating and cooling demands of
353 the buildings provides Fig. 7. Here, all 17 buildings are grouped into 6
354 building clusters (A – F). The building clusters are used in the investiga-
355 tions in Section 4. Buildings 1 and 2, a laboratory and a canteen, have the

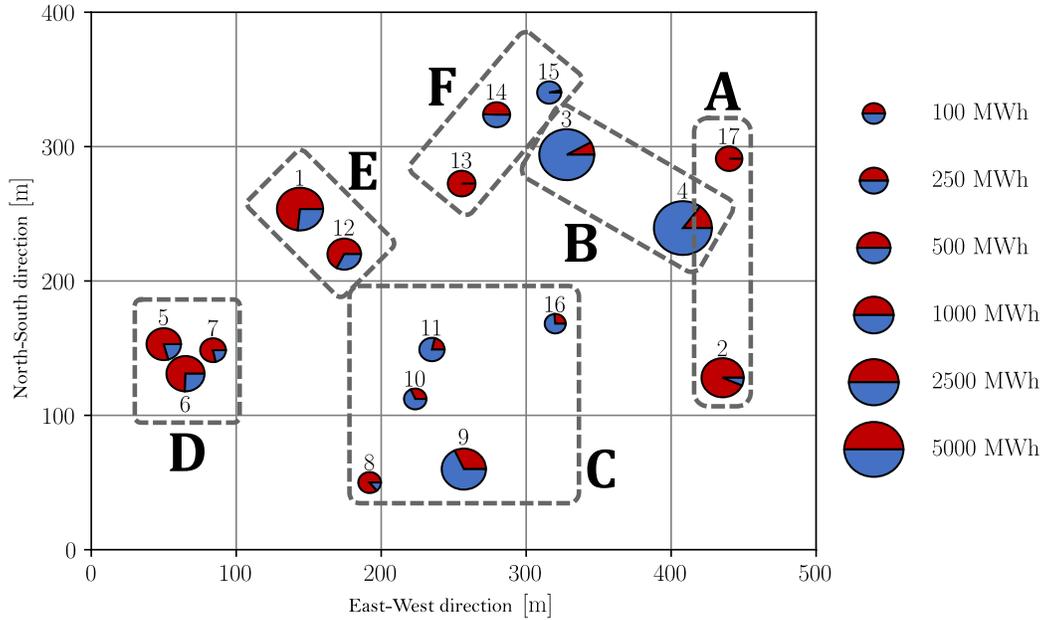


Figure 7: Total annual heating (red) and total annual cooling demand (blue) for 17 buildings on the research campus including two data centers (building 3 and 4). The building clusters A – F are used for generating different demand scenarios in Section 4. Illustration adapted from [35].

356 largest heating demands. These two buildings account for 49% of the heating
 357 demand (total heating demand: 6.36 GWh). Buildings 3 and 4 are data cen-
 358 ters, which account for 73% of the cooling demand (total cooling demand:
 359 10.0 GWh). Detailed building information is provided in Table A.4 in the
 360 Appendix. For the heating and cooling systems of the buildings, constant
 361 supply temperatures of $T_{h,sup} = 60\text{ }^{\circ}\text{C}$ and $T_{c,sup} = 16\text{ }^{\circ}\text{C}$ are assumed.

362 3.2. System evaluation

363 Based on the operation of the BLTN system, the four KPIs introduced in
 364 Section 2.2.2 are evaluated. The optimal configuration of the BLTN system

365 (determined with the linear program) is described in Appendix C. The key
 366 performance indicators for the BLTN system and the reference system are
 367 listed in Table 1 (data obtained from the optimization results). The specific
 368 supply costs of the BLTN solution are 37.6 EUR/MWh. The supply costs
 369 of the reference system are 11.9% higher (42.1 EUR/MWh). Moreover, the
 370 BLTN system performs substantially better from a thermodynamic perspec-
 371 tive: The exergy efficiency is 33.5%, and thus 6.8 percentage points higher
 372 compared to the reference system. The System COP of the BLTN system
 373 is 5.01, which means that with one unit of electric power 5 units of heating
 374 or cooling demand can be supplied. As a result of the higher efficiency, the
 375 CO₂ emissions of the BLTN system are lower (64.8 g/kWh) compared to the
 376 reference system (92.4 g/kWh).

Table 1: Comparison of KPIs of the system with BLTN and the reference system calculated based on the optimization results.

KPI	Unit	BLTN	Reference	
Specific supply costs	EUR/MWh	37.6	42.1	(+ 11.9%)
Exergy efficiency	—	33.5%	26.7%	(− 6.8 p.p.)
System COP	—	5.01	3.95	(− 1.06)
Specific CO ₂ emissions	g/kWh	64.8	92.4	(+ 42.6%)

377 3.3. Demand balancing

378 Based on (measured) heating and cooling demand profiles of all 17 build-
 379 ings, the District DOC is calculated to $\Phi_{\text{distr}} = 0.632$ (Eq. (6)). The cu-
 380 mulated thermal demands in different parts of the energy system are de-
 381 picted in Fig. 8. The total heating and cooling demands of the buildings

382 are 6359 MWh and 10042 MWh, respectively, which is illustrated in the left
 383 column in Fig. 8 a). The cumulated thermal demands of the building en-
 384 ergy systems (obtained from the optimization results) are illustrated in the
 385 second column: The heating demands of the building energy systems are
 386 lower (4556 MWh) than the original heating demands (6359 MWh) since the
 387 heat flow to the evaporator of heat pumps is smaller than the outgoing heat
 388 flow at the condenser. Furthermore, the operation of electric boilers lower
 389 the thermal demands of the BESs. The cooling demand (10042 MWh) re-
 390 mains the same since all cooling energy is provided by direct cooling (heat
 391 exchangers). Heating and cooling demands of BESs overlap and can be bal-
 392 anced to some extent: The proportion of balanced demands in buildings is
 393 expressed by the BES DOC (Eq. (10)). The BES DOCs of the 17 buildings
 394 (obtained from the optimization results) are listed in Table 2. They range
 395 between 0 and 0.52. The largest BES DOC is obtained in building 14, in
 396 which the cooling demand almost equals the heating demand, and thus more
 397 than half of the thermal demands are balanced in the building. Buildings 13
 398 and 17 do not have a cooling demand and therefore no balancing takes place
 399 ($\Phi_{\text{BES},13} = \Phi_{\text{BES},17} = 0$).

Table 2: Building energy system DOCs (Φ_{BES}) of all 17 buildings based on optimization results.

Building	1	2	3	4	5	6	7	8	9
Φ_{BES}	0.41	0.06	0.12	0.22	0.31	0.45	0.27	0.20	0.50
Building	10	11	12	13	14	15	16	17	
Φ_{BES}	0.24	0.33	0.45	0	0.52	0.05	0.37	0	

400 Based on Eq. (11), a mean BES DOC of $\overline{\Phi_{\text{BES}}} = 0.251$ is obtained and

3668 MWh are balanced in buildings (c.f. Fig. 8 a)). As a result, the remaining heating demand, which is covered by the BLTN, is 2722 MWh and the cooling demand 8208 MWh. The sum of the remaining heating and cooling demands is

$$14598 \text{ MWh} \cdot (1 - \bar{\Phi}_{\text{BES}}) = 10930 \text{ MWh} \quad (26)$$

as illustrated in Fig. 8 (b).

The proportion of demands that is balanced in the BLTN is quantified by the Network DOC. With Eq. (15), the Network DOC is calculated to $\Phi_{\text{netw}} = 0.457$ (based on the optimization results). Due to network balancing, the heating demand that needs to be covered by the energy hub is further reduced from 2722 MWh to 224 MWh. Accordingly, the cooling demand covered by the energy hub is reduced from 8208 MWh to 5710 MWh, as shown in Fig. 8 a). The sum of the heating and cooling demands which needs to be covered by the energy hub is then

$$10930 \text{ MWh} \cdot (1 - \Phi_{\text{netw}}) = 5934 \text{ MWh} \quad (27)$$

In this balance, heat losses (or gains) of the network are neglected since they play a minor role: The net heat loss of the warm pipe is 6 MWh and the net thermal loss of the cold pipe is 29 MWh.

3.3.1. *Balancing over the course of the year*

The DOC is an aggregated metric which does not reveal any information about the temporal distribution of the demand balancing. To investigate the temporal distribution, the demand balancing is illustrated for the considered year in Fig. 9. The cumulated demands of the district (measured data) are depicted Fig. 9 a): The cooling demand occurs throughout the year and

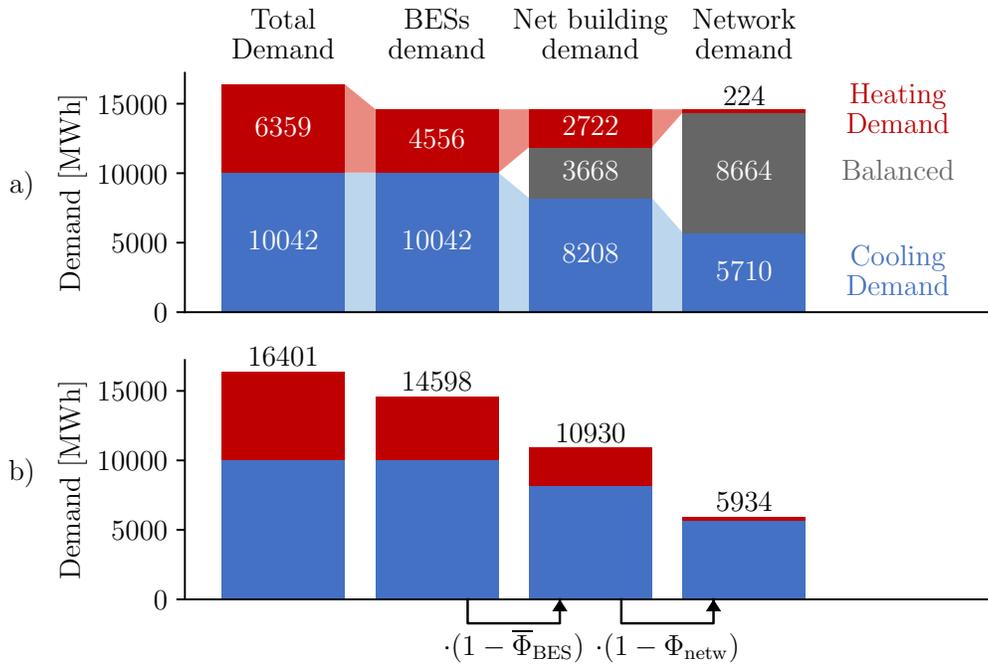


Figure 8: Subfigure a) illustrates from left to right: The total heating and cooling demands of all buildings in the district (6359 MWh/10042 MWh); the thermal demands of the building energy systems (BESs); the thermal demand transferred from the BLTN to the buildings (3668 MWh are balanced in buildings); the residual demand of the BLTN that is covered by the energy hub (4996 MWh are balanced in the BLTN). Subfigure b) illustrates the sum of the heating and cooling demands depicted in Subfigure a).

423 reaches its peak in summer. The heating demand predominantly occurs
 424 during winter months. A small heating demand occurs also during summer.
 425 In Fig. 9 b), the thermal demands of the BESs (based on the optimization
 426 results) are illustrated (denoted by $\dot{Q}_{h,BES,b,t}/\dot{Q}_{c,BES,b,t}$ in Fig. 3). The change
 427 of the heating demands from Fig. 9 a) to b) results from heat pumps as
 428 well as the electric boilers and heat storages in the BESs (balancing not yet
 429 included): The heat demand of the BESs is lower since the heat flow at
 430 the evaporator of heat pumps is lower than the outgoing heat flow at the
 431 condenser. Peak demands, e.g. in the second half of January, are shaved
 432 with electric boilers and thermal storages. The heating demand profile in
 433 Fig. 9 b) shows a maximum of 1.1 MW. In Fig. 9 c), the net demand of the
 434 buildings (after balancing in buildings) is depicted. The change of demands
 435 from Fig. 9 b) to c) is due to the balancing in buildings. The peak heating
 436 demand is lowered from 1.1 to 0.8 MW and the peak cooling demand to
 437 2.2 MW. Fig. 9 d) depicts the thermal demands which are not balanced in
 438 the network, and are covered by the energy hub. From Fig. 9 c) to d), the
 439 thermal demands reduce due to balancing between buildings. As illustrated
 440 in Fig. 8, the remaining heating demand is 224 MWh and the remaining
 441 cooling demand is 5710 MWh. Due to the balancing in buildings and network,
 442 heating demands are almost completely canceled out. The peak heating
 443 demand is reduced from 1.88 MW to 0.49 MW (-74%) and the peak cooling
 444 demand (begin of July) is reduced from 2.43 MW to 2.25 MW (-7%). During
 445 winter, heating and cooling demands are balanced out almost completely.
 446 During summer, a large residual cooling demand remains that is covered by
 447 the energy hub.

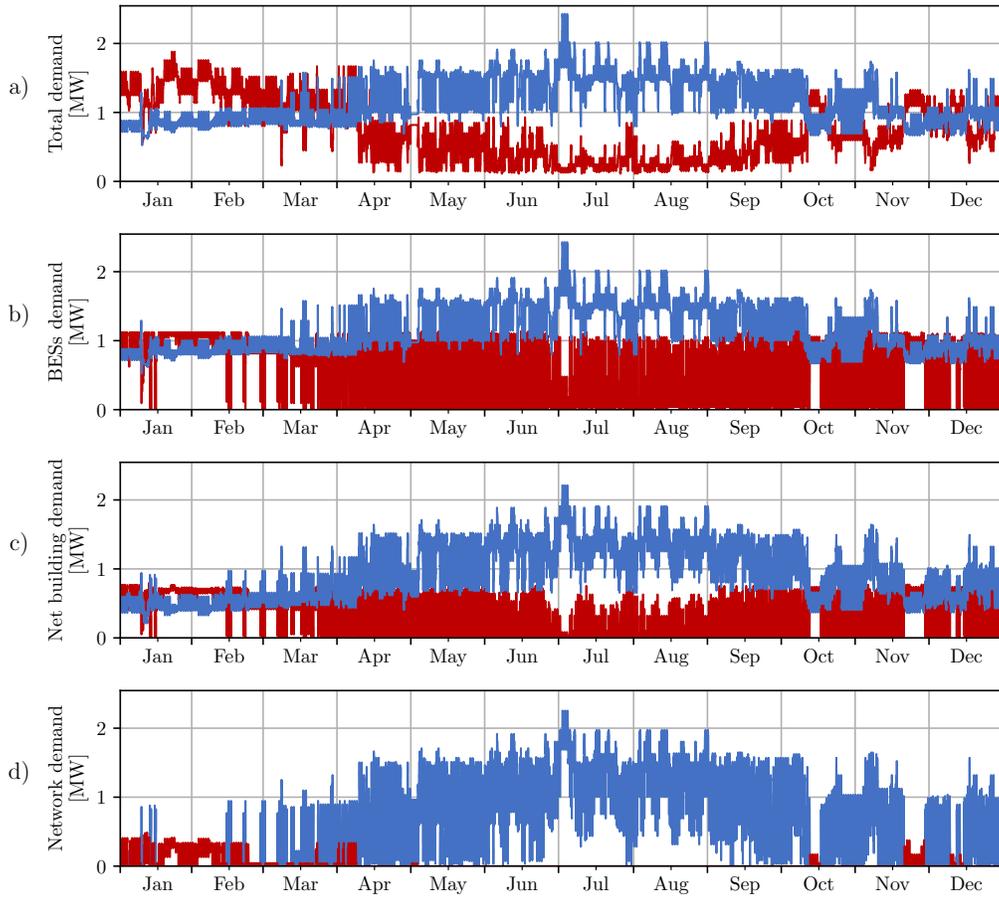


Figure 9: Cumulated heating and cooling demands of all buildings for one year: a) Total building demand (for space heating and cooling), b) thermal demand of building energy systems (BESs) before balancing in buildings, c) net building demands (after balancing in buildings), d) residual network demand that is covered by the energy hub (after balancing between buildings).

448 4. Correlation of DOC and key performance indicators

449 In this section, the performance and profitability of district energy sys-
450 tems with BLTN are related to the demand structure, especially the simul-
451 taneity of heating and cooling demands. In order to investigate this corre-
452 lation, demand scenarios are derived from the real-world use case and for
453 each of these scenarios the performance of a BLTN is evaluated. Section 4.1
454 presents the methodology for generating different demand scenarios. In Sec-
455 tion 4.2, correlations between different DOCs are investigated. Section 4.3
456 presents correlations between the system performance and the demand struc-
457 ture as well as a comparison with the reference system.

458 4.1. Demand scenario generation

459 By selecting different subsets of buildings of the original use case (de-
460 scribed in Section 3), a large variety of different demand scenarios is gen-
461 erated. For this purpose, the 17 buildings are aggregated into 6 building
462 clusters (A, B, C, D, E, F), as depicted in Fig. 7. In Table 3, the demands of
463 all building clusters are listed along with their District DOC. Furthermore,
464 a heating-cooling ratio $R \in [-1, 1]$ is considered, which indicates whether
465 heating or cooling demands are predominant in the district:

$$R = \frac{Q_{h,dem}^{tot} - Q_{c,dem}^{tot}}{Q_{h,dem}^{tot} + Q_{c,dem}^{tot}} \quad (28)$$

466 $R = 1$ indicates a scenario with only heating demands, $R = -1$ means only
467 cooling demands are observed. If heating demands equal cooling demands,
468 the demand ratio is $R = 0$.

469 Based on the 6 building clusters, $2^6 - 1 = 63$ non-empty cluster subsets
470 can be selected. In each of the 63 subsets, different building clusters are

Table 3: Building clusters (A-F) used for demand scenario generation.

	Buildings	$\dot{Q}_{h,dem}^{tot}$	$\dot{Q}_{c,dem}^{tot}$	$\dot{Q}_{h,dem}^{peak}$	$\dot{Q}_{c,dem}^{peak}$	Φ_{distr}	R
		[MWh]	[MWh]	[MW]	[MW]	—	—
A	1, 12	1410	84	0.48	0.33	0.05	0.89
B	2, 17	987	7347	0.33	1.41	0.24	-0.76
C	3, 4	677	1336	0.28	0.41	0.61	-0.33
D	5, 6, 7	1233	377	0.37	0.15	0.38	0.53
E	8, 9, 10, 11, 16	1702	658	0.52	0.28	0.44	0.44
F	13, 14, 15	350	241	0.14	0.06	0.65	0.19

471 selected and, as a result, 63 different demand scenarios are considered. Each
 472 demand scenario has an individual demand structure and District DOC. The
 473 performance of a BLTN system is evaluated for each demand scenario with
 474 the optimization model introduced in Section 2.2.1.

475 4.2. Correlation between different DOCs

476 In this section, a correlation between the District DOC and the Network
 477 DOC/mean BES DOC is derived. The District DOC is calculated with the
 478 measured demand profiles, the Network DOC and mean BES DOC is ob-
 479 tained from the optimization results. In Fig. 10, the three DOCs are plotted
 480 for all 63 demand scenarios: The Network DOC is plotted against the mean
 481 BES DOC and the District DOC is indicated by the coloring. Except for one
 482 demand scenario, the Network DOC ranges between 0 and 0.5. Similarly, the
 483 mean BES DOC does not exceed 0.5. No correlation between the Network
 484 DOC and mean BES DOC is observed.

485 However, a correlation between the total share of balanced demands (ex-

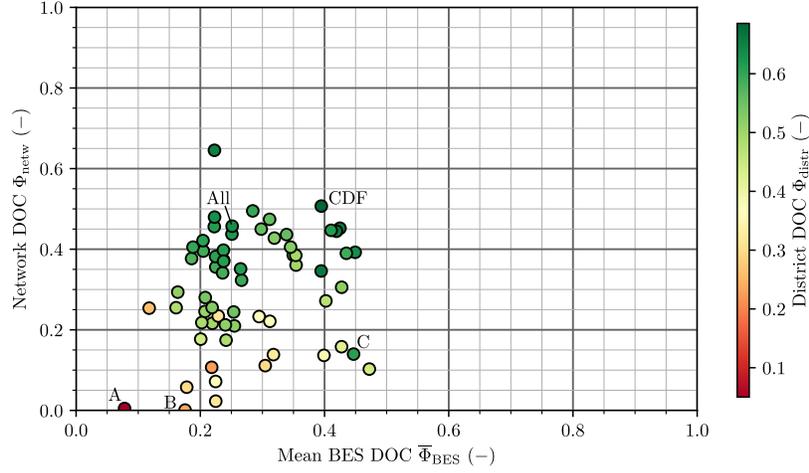


Figure 10: Illustration of the three Demand Overlap Coefficients: Network DOC, mean BES DOC and District DOC.

486 pressed with the mean BES DOC and Network DOC) and the District DOC
 487 is found, as illustrated in Fig. 11. On the horizontal axis, the product
 488 $(1 - \Phi_{\text{netw}})(1 - \bar{\Phi}_{\text{BES}})$ is plotted, which is the proportion of demands that can-
 489 not be balanced in buildings or BLTN (c.f. Eq. (18)). The term $(1 - \Phi_{\text{distr}})$ is
 490 plotted on the vertical axis. A distinctive correlation is observed, which shows
 491 that the total demand balancing (in buildings and network) can be derived
 492 approximately from the District DOC: $(1 - \Phi_{\text{netw}})(1 - \bar{\Phi}_{\text{BES}}) \approx (1 - \Phi_{\text{distr}})$.
 493 The District DOC Φ_{distr} is calculated solely with the building's heating and
 494 cooling demands (Eq. (6)) and no detailed knowledge about the system de-
 495 sign or operation is required. Therefore, relations between the District DOC
 496 and the system performance are investigated in the following section.

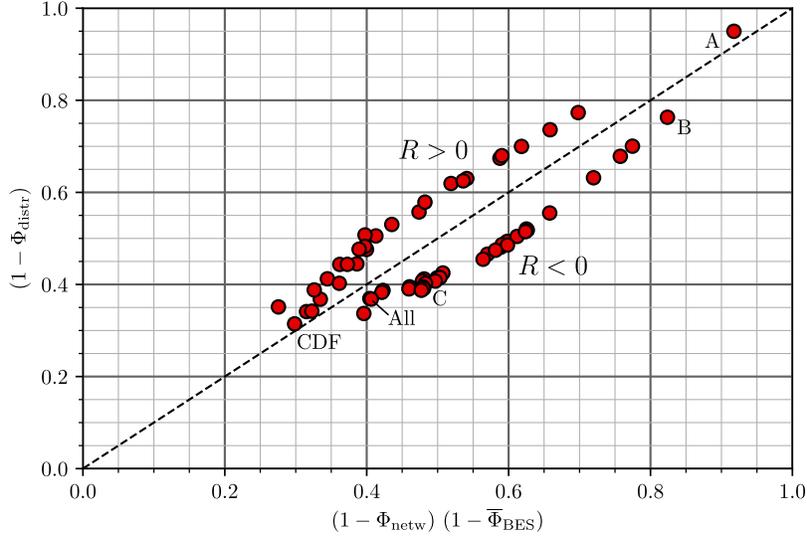


Figure 11: The share of thermal demands, that cannot be balanced in buildings or BLTN, i.e. $(1 - \Phi_{\text{netw}})(1 - \bar{\Phi}_{\text{BES}})$, correlates with the District DOC Φ_{distr} .

497 4.3. Correlation between system performance and District DOC

498 In this section, correlations between the District DOC and key perfor-
 499 mance indicators (obtained from optimization results) are investigated. In
 500 Fig. 12, the specific supply costs are plotted against the District DOC for all
 501 63 demand scenarios. The color indicates the heating-cooling ratio R . The
 502 supply costs of demand scenarios with a larger heating than cooling demands
 503 ($R > 0$) decrease with increasing District DOC. The demand scenario, which
 504 is labeled *All*, comprises all building clusters and is therefore identical to the
 505 case study investigated in Section 3. The highest supply costs are observed
 506 in demand scenario A. In this scenario, heating demands dominate and al-
 507 most no demand balancing takes place. For demand scenarios with larger
 508 District DOC, more and more waste heat is recovered (from buildings with

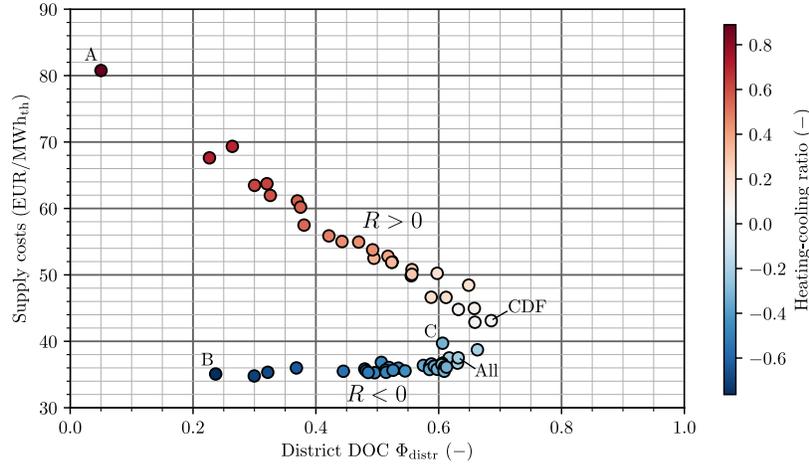


Figure 12: Specific heating and cooling supply costs of all 63 demand scenarios are plotted against the District DOC. The demand scenario comprising all building clusters is labeled *All*.

509 cooling demands). This reduces the operation of the reversible heat pump
510 in the energy hub, and thus electricity costs. In Fig. 12, scenario CDF has
511 the largest DOC ($\Phi_{\text{distr}} \approx 0.65$) and the total heating and cooling demands
512 are almost equal ($R \approx 0$). For demand scenarios in which cooling demands
513 dominate, specific supply costs are almost constant. This is explained as
514 follows: On the one hand, the lower the heating demands, the lower the spe-
515 cific supply costs, since cooling demands can be covered at lower costs than
516 heating demands. (This results from the fact, that for direct cooling with
517 the cold pipe of the BLTN no electricity is needed). On the other hand, the
518 lower the heating demands, the higher the load of the reversible heat pump
519 in the energy hub (in order to balance the residual cooling demands of the
520 BLTN), which causes additional electricity costs. The two effects cancel each
521 other out.

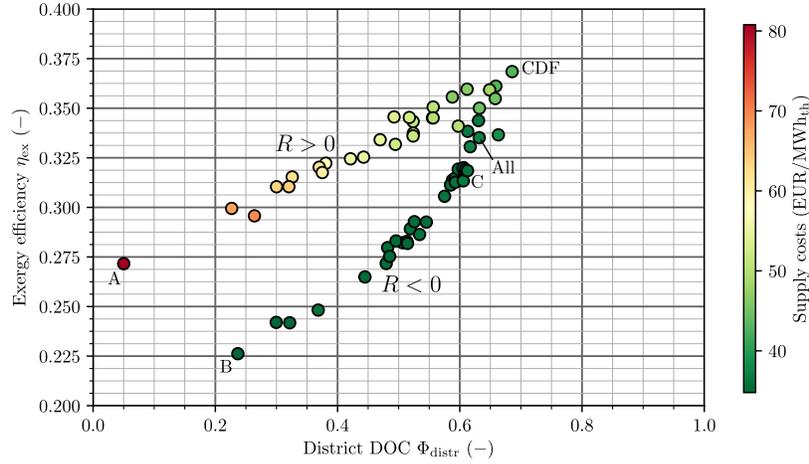


Figure 13: Exergy efficiency plotted against District DOC. The exergy efficiency correlates in both branches ($R > 0$ and $R < 0$) with the District DOC: Larger District DOCs result in higher exergy efficiencies.

522 In Fig. 13, the exergy efficiency is plotted against the District DOC for
 523 all demand scenarios. Similar to Fig. 12, two branches with positive and neg-
 524 ative heating-cooling ratios are observed. The exergy efficiency of demand
 525 scenarios in both branches correlate positively with the District DOC. De-
 526 mand scenario CDF, which has the largest District DOC, also has the largest
 527 exergy efficiency (36.8%). Demand scenarios with a positive heating-cooling
 528 ratio show a larger exergy efficiency. This results from the fact that in the
 529 calculation of the exergy efficiency, benefits from covered heating demands
 530 (heat flow at 60 °C) are weighted more heavily than covered cooling demands
 531 (heat flow close to ambient temperature). The coloring in Fig. 13 indicates
 532 the specific supply costs. It is remarkable that exergy efficiency only corre-
 533 lates with supply costs for demand scenarios with $R > 0$. However, a general
 534 correlation between exergy efficiency and supply costs cannot be observed.

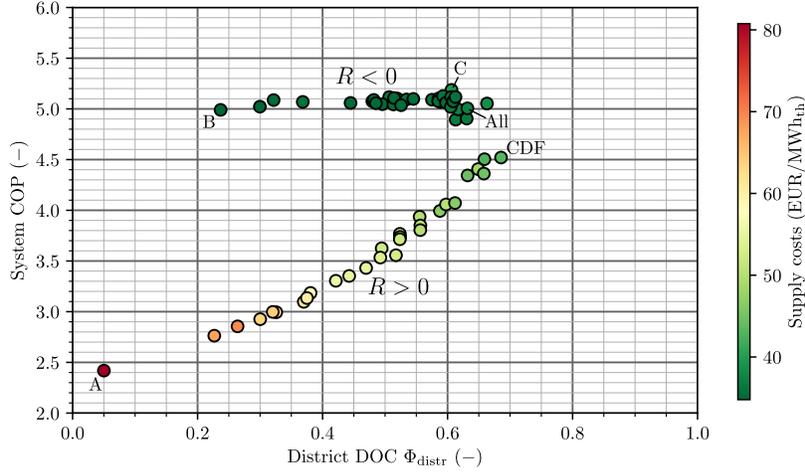


Figure 14: System coefficient of performance plotted against District DOC: For scenarios with more heating demands than cooling demands, the System COP increases with larger District DOCs. A correlation between System COP and supply costs is observed.

535 In Fig. 14, the System COP is plotted against the District DOC. In
 536 contrast to the exergy efficiency, a strong correlation between System COP
 537 and supply costs can be observed. This is due to the fact that, in terms
 538 of benefit, heating and cooling demands are weighted equally in the System
 539 COP (Eq. (24)) as well as in the specific supply costs (Eq. (20)).

540 4.3.1. Comparison with reference system

541 In this section, the performance of a district energy system with BLTN is
 542 compared with the performance of the reference system (c.f. Section 2.2.3)
 543 for all 63 demand scenarios. Intervals of the District DOC are identified for
 544 which the BLTN system performs better than the reference system.

545 In Fig. 15, the specific supply costs and the District DOC are depicted
 546 for the BLTN system and the reference system. The two branches ($R >$

547 0 and $R < 0$) are identified for the reference system as well. For $R >$
548 0 (heating demand larger than cooling demand), the supply costs of the
549 reference system decrease with increasing District DOC. This results from
550 the fact that cooling demands can be covered at lower costs compared to
551 heating demands: The COP of chillers always exceeds 5, whereas the COP of
552 air source heat pumps ranges between 2.9 and 4.5 (depending on the ambient
553 air temperature). Therefore, the supply costs decrease for an increasing share
554 of cooling demands. The overall trend for both systems (BLTN and reference
555 system) is the same. However, the offset and slope of the trend lines are
556 different. For demand scenarios with a small District DOC ($\Phi_{\text{distr}} < 0.4$), the
557 supply costs of the BLTN system are higher than the reference system. On
558 the other hand, for large District DOCs ($\Phi_{\text{distr}} > 0.45$), the BLTN system
559 has lower supply costs than the reference system. This results from cost
560 savings in the BLTN system with increasing demand balancing potential. A
561 high District DOC indicates that a large proportion of heating demands can
562 be supplied by waste heat from cooling applications. As a result, the overall
563 efficiency increases and the supply costs decrease. In the reference system,
564 a large overlap of heating and cooling demands does not lead to a higher
565 overall efficiency since heating and cooling demands are covered separately
566 from each other. As a result, BLTN systems become more efficient with
567 larger District DOCs.

568 This effect is also reflected by the exergy efficiencies. Fig. 16 shows the
569 exergy efficiency of all demand scenarios. For the BLTN system, the demand
570 scenario with the largest District DOC has the highest exergy efficiency. The
571 exergy efficiency of the reference system ranges between 24 % and 31 %, and

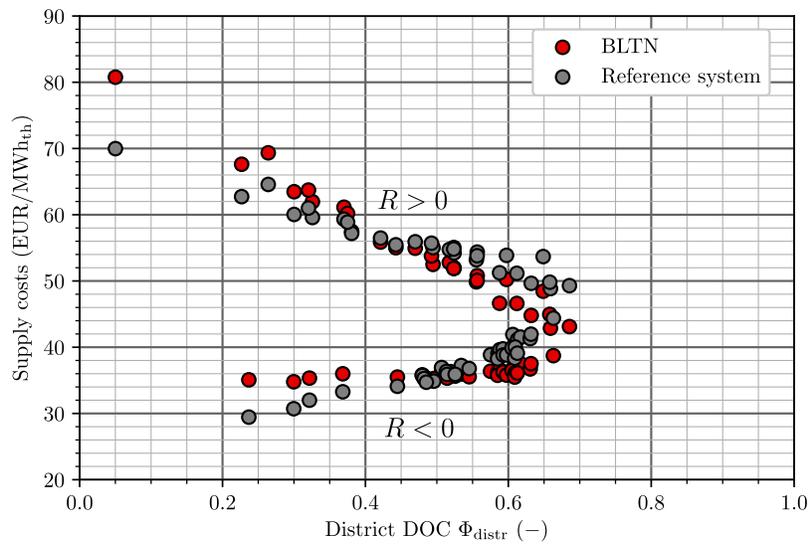


Figure 15: Supply costs and District DOC of the BLTN system and the reference system. For large District DOCs ($\Phi_{\text{distr}} > 0.45$), the system with BLTN has lower specific supply costs than the reference system.

572 thus varies less with different District DOCs. The reference system has a
573 larger exergy efficiency than the BLTN system for District DOCs smaller
574 than 0.25. For District DOCs larger than 0.3, the exergy efficiency of the
575 BLTN system is higher than the reference system. This is in line with the
576 findings by Pass et al. [34], who show that for the special case of a (static)
577 demand situation with at least 1 unit cooling per 5.7 units of heating demand,
578 a BLTN is more exergy efficient than a supply with decentral building energy
579 systems (heat pumps and direct cooling). A (static) demand ratio of 1:5.7
580 equals a DOC of 0.298 and therefore affirms our findings (DOC = 0.3). The
581 substantial increase of the exergy efficiency of the BLTN system with larger
582 District DOCs is a result of the increased demand balancing potential. For
583 large District DOCs, a large proportion of waste heat from chillers can be
584 used as heat source for heat pumps and less thermal energy must be provided
585 by the energy hub.

586 **5. Conclusions**

587 The efficiency and profitability of district energy systems with BLTN
588 strongly depend on the heating and cooling demand structure of the con-
589 nected buildings. BLTNs are best suited when heating and cooling demands
590 are of the same magnitude and occur simultaneously. In this paper, the DOC
591 is introduced which quantifies the simultaneity of heating and cooling de-
592 mands. The DOC can be calculated for a district (District DOC), a building
593 energy system (BES DOC) and a thermal network (Network DOC). Based
594 on the use case, 63 demand scenarios are generated in order to investigate
595 correlations between DOCs and key performance indicators.

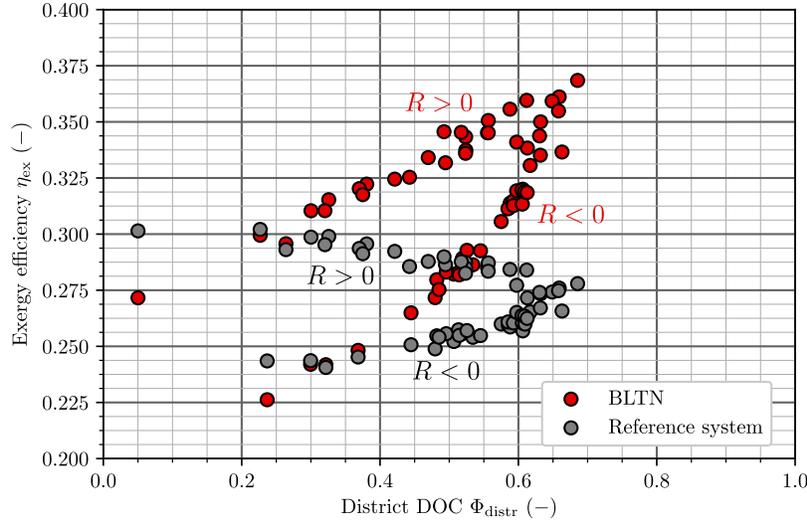


Figure 16: Exergy efficiency and District DOC of the system with BLTN and the reference system. For demand scenarios with a District DOC larger than 0.3, the exergy efficiency of the system with BLTN is higher than in the reference system.

596 A distinctive correlation between the Network DOC, mean BES DOC
 597 and District DOC is observed, from which the approximation $(1 - \Phi_{\text{netw}})(1 -$
 598 $\bar{\Phi}_{\text{BES}}) \approx (1 - \Phi_{\text{distr}})$ is derived. This means, the District DOC alone is suffi-
 599 cient to estimate which proportion of demands can be balanced in buildings
 600 and the BLTN. This is an important finding since for the calculation of the
 601 District DOC no detailed knowledge about the network or building energy
 602 systems is needed. The District DOC is calculated solely with the buildings'
 603 heating and cooling demands which usually are available in an early plan-
 604 ning phase of district energy systems. This makes the District DOC a widely
 605 applicable key metric in the planning process.

606 The analysis of the demand scenarios shows that the District DOC cor-
 607 relates with the system's exergy efficiency: The demand scenario with the

608 largest District DOC has the highest exergy efficiency. For District DOCs
609 larger than 0.3, a heating and cooling supply with a BLTN has a higher
610 exergy efficiency compared to a reference system. This is in line with the
611 findings by Pass et al. [34]. Moreover, district energy systems with BLTN
612 have lower specific supply costs than a state-of-the-art reference system if the
613 District DOC exceeds 0.45. The better economic and thermodynamic perfor-
614 mance of systems with large District DOC is a result of the larger potential
615 for balancing demands in the districts.

616 In summary, this study shows the importance of the district’s demand
617 structure for planning district energy systems and BLTN systems in partic-
618 ular. The District DOC turns out to be a meaningful metric which allows
619 to characterize the demand structure of a district. Thus, the District DOC
620 helps to identify clusters of buildings with complementary heating and cool-
621 ing demand profiles and to decide which buildings should be connected to a
622 BLTN.

623 In future works, the DOC metric can be extended in order to consider
624 balancing effects that are achieved by central thermal storages in a BLTN.
625 Calculating a DOC for a central storage can help to quantify to what extend
626 the demand balancing can be increased by installing the storage, and to
627 determine an optimal storage capacity as it is a trade-off between investment
628 and demand balancing potential. In addition, heat losses of the network can
629 be included in the DOC calculation as they also contribute to the balancing
630 of heating and cooling demands.

631 **6. Acknowledgments**

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633 Initiative “Energy System 2050 – A Contribution of the Research Field En-
634 ergy”.

635 **7. Nomenclature**

636 **Abbreviations**

5GDHC	5th generation district heating and cooling
ASHP	Air source heat pump
BAT	Battery
BES	Building energy system
BLTN	Bidirectional low temperature network
CC	Compression chiller
DOC	Demand overlap coefficient
EH	Energy hub
637 HP	Heat pump
ISHP	Ice storage heat pump
ITES	Ice thermal energy storage
KPI	Key performance indicator
O&M	Operation and maintenance
TAC	Total annualized costs
TES	Thermal energy storage
638 PV	Photovoltaics

639 **Indices and Sets**

640	$b \in B$	Buildings
641	$t \in T$	Time steps
642	Variables	
	c	Specific supply costs
	COP_{Sys}	System COP
	e	CO ₂ emissions
	P	Electric power
643	Q	Thermal energy
	\dot{Q}	Thermal power
	T	Temperature
	W	Electric energy
644	Φ	Demand Overlap Coefficient
645	Parameters	
	kA	Heat loss coefficient
	R	Demand ratio
646	COP	Coefficient of performance
647	η	Efficiency
648	Subscripts	

bal	balancing
c	cooling
dem	demand
distr	district
h	heating
netw	network
649 ref	reference
res	residual
ret	return
sup	supply
th	thermal
650 tot	total

651 **Appendix A. Building data**

652 Detailed data about the 17 buildings is provided in Table A.4.

653 **Appendix B. Linear program**

654 The linear program is based on an optimization model by Wirtz et al. [35].
655 In this section, all model differences compared to the formulation in [35] are
656 presented in detail. All model parameters which are not presented in [35] or
657 have been modified are listed in Appendix B.4.

658 *Appendix B.1. Objective function*

659 The objective function are total annualized costs (TAC), which are based
660 on VDI 2067 [40] and include annualized investments of the equipment of the

661 energy hub (C_{EH}) and the building energy systems (C_{BES}), electricity costs
 662 (C_{el}) and revenues from electricity feed-in ($R_{\text{feed-in}}$) as well as investment for
 663 the thermal network (C_{netw}):

$$TAC = C_{\text{EH}} + C_{\text{BES}} + C_{\text{el}} - R_{\text{feed-in}} + C_{\text{netw}} \quad (\text{B.1})$$

664 The definitions of the cost proportions do not differ from the ones presented
 665 in [35].

666 *Appendix B.2. Building energy system (BES)*

667 The constraints of the building energy system remain unchanged except
 668 for the following adaptations: Firstly, it is assumed that the heat storage is
 669 ideally stratified. The heat pump (with a supply temperature of 60 °C) can
 670 charge the storage regardless of its state of charge and heat from the heat
 671 storage can always be used to cover the buildings heat demand (at 60 °C).
 672 Therefore the following constraint from the original formulation is omitted
 673 without substitution:

$$\dot{Q}_{\text{h,EB},b,d,t} \geq \dot{Q}_{\text{h,TES},b,d,t}^{\text{ch}} \quad \forall b \in \text{B}, d \in \text{D}, t \in \text{T} \quad (\text{B.2})$$

674 $\dot{Q}_{\text{h,EB},b,d,t}$ denotes the heat output of the electric boiler (EB) and $\dot{Q}_{\text{h,TES},b,d,t}^{\text{ch}}$
 675 the heat flow charging the heat storage. The storage is assumed to be charged
 676 and discharged directly without additional heat exchangers. Therefore, all
 677 losses related to the charging and discharging process are neglected ($\eta_{\text{TES}}^{\text{ch}} =$
 678 $\eta_{\text{TES}}^{\text{dch}} = 1$).

679 Furthermore, some constraints are simplified due to the absence of cooling
 680 towers in the building energy systems. In particular, the cooling balance of

681 a building is reduced to

$$\dot{Q}_{c,CC,b,d,t} + \dot{Q}_{c,DRC,b,d,t} = \dot{Q}_{c,dem,b,d,t} \quad \forall b \in B, d \in D, t \in T \quad (B.3)$$

682 Here, $\dot{Q}_{c,CC,b,d,t}$ denotes the cooling power of the compression chiller (CC)
 683 and $\dot{Q}_{c,DRC,b,d,t}$ the cooling power of the direct cooler (DRC).

684 *Appendix B.3. Energy hub (EH)*

685 The superstructure of the energy hub is changed substantially compared
 686 to the formulation in [35]. Therefore, all constraints for the energy hub are
 687 explained in detail.

688 **Generation units and storages**

689 The thermal or electric power of the units is limited by their rated power:

$$\dot{Q}_{h,k,EH,d,t} \leq \dot{Q}_{h,k,EH}^{\text{nom}} \quad \forall k \in \{\text{ASHP}, \text{ISHP}\}, d \in D, t \in T \quad (B.4)$$

$$\dot{Q}_{c,ASHP,EH,d,t} \leq \dot{Q}_{c,ASHP,EH}^{\text{nom}} \quad \forall d \in D, t \in T \quad (B.5)$$

$$P_{PV,EH,d,t} \leq P_{PV,EH}^{\text{nom}} \quad \forall d \in D, t \in T \quad (B.6)$$

690 The capacity of the reversible air source heat pump $cap_{\text{ASHP,EH}}$ which is
 691 needed to calculate the investment is defined by the constraints

$$\dot{Q}_{h,ASHP,EH}^{\text{nom}} \leq cap_{\text{ASHP,EH}} \quad (B.7)$$

$$\dot{Q}_{c,ASHP,EH}^{\text{nom}} \leq cap_{\text{ASHP,EH}} \quad (B.8)$$

692 The total module area of PV (A_{PV}) is limited by the maximum available
 693 area:

$$A_{PV} \leq A_{PV}^{\text{max}} \quad (B.9)$$

694 The rated power of the PV modules is

$$P_{PV,EH}^{\text{nom}} = G_{\text{sol,STC}} A_{PV} \eta_{\text{PV,STC}} \quad (B.10)$$

695 Here, $G_{\text{sol,STC}}$ denotes the global tilted irradiance and $\eta_{\text{PV,STC}}$ the electric
 696 efficiency under Standard Test Conditions. The power of the PV modules is

$$P_{\text{PV,EH},d,t} \leq G_{\text{sol},d,t} A_{\text{PV}} \eta_{\text{PV},d,t} \quad \forall d \in \text{D}, t \in \text{T} \quad (\text{B.11})$$

697 For energy conversion units constant or time-dependent efficiencies are
 698 assumed. For the air source heat pump (ASHP), a heating and cooling COP
 699 ($\text{COP}_{\text{h,ASHP},d,t} / \text{COP}_{\text{c,ASHP},d,t}$) is calculated a priori with the ambient air
 700 temperature. The input-output constraints are

$$\dot{Q}_{\text{h,ASHP,EH},d,t} = P_{\text{h,ASHP,EH},d,t} \text{COP}_{\text{h,ASHP},d,t} \quad \forall d \in \text{D}, t \in \text{T} \quad (\text{B.12})$$

$$\dot{Q}_{\text{c,ASHP,EH},d,t} = P_{\text{c,ASHP,EH},d,t} \text{COP}_{\text{c,ASHP},d,t} \quad \forall d \in \text{D}, t \in \text{T} \quad (\text{B.13})$$

$$P_{\text{ASHP,EH},d,t} = P_{\text{h,ASHP,EH},d,t} + P_{\text{c,ASHP,EH},d,t} \quad \forall d \in \text{D}, t \in \text{T} \quad (\text{B.14})$$

701 $P_{\text{ASHP,EH},d,t}$ denotes the total power demand of the air source heat pump.
 702 For discharging the ice thermal energy storage (ITES), an ice storage heat
 703 pump (ISHP) has to be installed, which freezes the fluid in the storage.
 704 The efficiency of the ISHP ($\text{COP}_{\text{h,ISHP},d,t}$) is calculated a priori based on the
 705 storage temperature (assumed 0 °C) and the temperature of the warm pipe
 706 of the BLTN:

$$\dot{Q}_{\text{h,ISHP,EH},d,t} = P_{\text{ISHP,EH},d,t} \text{COP}_{\text{h,ISHP},d,t} \quad \forall d \in \text{D}, t \in \text{T} \quad (\text{B.15})$$

$$\dot{Q}_{\text{h,ISHP,EH},d,t} = \dot{Q}_{\text{h,ITES,EH},d,t}^{\text{dch}} + P_{\text{ISHP,EH},d,t} \quad \forall d \in \text{D}, t \in \text{T} \quad (\text{B.16})$$

707 The ice storage is charged by passing water from the warm pipe of the BLTN
 708 through the pipes of the ice storage (no auxiliary power needed).

709 Storages are modeled with a formulation that allows a seasonal operation
 710 (as presented by [41] and [42]) and in accordance with the formulation in

711 [35]. The ice thermal energy storage is modeled in the same manner except
 712 for minor changes: The standby losses of the ice storage do not depend on its
 713 state of charge since the temperature is assumed constant (0 °C). Therefore,
 714 losses only depend on the storage capacity (i.e. storage surface area):

$$S_{\text{ITES,EH}}^{\text{cap}} \phi_{\text{ITES,EH,loss}} \quad (\text{B.17})$$

715 Furthermore, the charging power is limited by a fixed value instead of a
 716 capacity share:

$$\dot{Q}_{\text{c,ITES,EH},d,t}^{\text{ch}} \leq \dot{Q}_{\text{c,ITES,EH}}^{\text{ch,max}} \quad \forall d \in \text{D}, \quad t \in \text{T} \quad (\text{B.18})$$

717 The discharging power of the ITES is constrained by the operation limits of
 718 the ice storage heat pump according to Eq. (B.16).

719 Energy balances

720 The thermal balance of the energy hub is:

$$\begin{aligned} & \dot{Q}_{\text{h,ASHP,EH},d,t} + \dot{Q}_{\text{h,ISHP,EH},d,t} - \dot{Q}_{\text{c,ASHP,EH},d,t} \\ & - \dot{Q}_{\text{c,ITES,EH},d,t}^{\text{ch}} = \dot{Q}_{\text{res,EH},d,t} + \dot{Q}_{\text{netw},d,t} \\ & \forall d \in \text{D}, \quad t \in \text{T} \end{aligned} \quad (\text{B.19})$$

721 Here, $\dot{Q}_{\text{h,ASHP,EH},d,t}$ and $\dot{Q}_{\text{h,ISHP,EH},d,t}$ describe the heat from the ASHP and
 722 ISHP, respectively. $\dot{Q}_{\text{c,ASHP,EH},d,t}$ denotes the cooling power of the ASHP and
 723 $\dot{Q}_{\text{c,ITES,EH},d,t}^{\text{ch}}$ denotes the thermal charging power of the ice storage. $\dot{Q}_{\text{res,EH},d,t}$
 724 and $\dot{Q}_{\text{netw},d,t}$ denote the residual heat demand of all buildings (including
 725 thermal network losses) and the heat needed to raise or lower the temperature

726 of the BLTN, respectively. The electricity balance of the energy hub is

$$\begin{aligned}
P_{\text{PV,EH},d,t} + P_{\text{grid},d,t} + P_{\text{BAT,EH},d,t}^{\text{dch}} &= \sum_{b \in B} P_{\text{BES},b,d,t} + P_{\text{pumps},d,t} \\
+ P_{\text{ASHP,EH},d,t} + P_{\text{ISHP,EH},d,t} + P_{\text{feed-in},d,t} + P_{\text{BAT,EH},d,t}^{\text{ch}} \\
\forall d \in D, t \in T & \tag{B.20}
\end{aligned}$$

727 Here, $\sum_{b \in B} P_{\text{BES},b,d,t}$ denotes the cumulated power demand by heat pumps,
728 compression chillers and electric boilers in the buildings. $P_{\text{grid},d,t}/P_{\text{feed-in},d,t}$
729 denote the electric energy taken from and fed into the public power grid,
730 respectively. The electric demands of the hydraulic pumps $P_{\text{pumps},d,t}$ are
731 calculated a priori as described in [35].

732 *Appendix B.4. Model parameters*

733 In this section, all model parameters which are not presented in [35] or
734 have been modified compared to the original formulation are listed. The heat
735 pump COPs (ASHP and ISHP) are calculated with the Carnot efficiency:

$$COP_{\text{h}} = \eta_{\text{Carnot}} COP_{\text{Carnot}} = \eta_{\text{Carnot}} \frac{T_{\text{sink}}}{T_{\text{sink}} - T_{\text{source}}} \tag{B.21}$$

736 Similarly, for the cooling mode of the ASHP

$$COP_{\text{c}} = \eta_{\text{Carnot}} \frac{T_{\text{source}}}{T_{\text{sink}} - T_{\text{source}}} \tag{B.22}$$

737 is applied. In Eq. (B.21) and (B.22), T_{sink} and T_{source} denote the condensing
738 temperature and evaporating temperature of the refrigerant, respectively.
739 For each heat transfer, a minimal temperature difference ΔT^{min} between the
740 two sides of the heat exchanger is considered. $\Delta T^{\text{min}} = 2 \text{ K}$ is applied in case
741 of water to water heat transfer and $\Delta T^{\text{min}} = 10 \text{ K}$ in case of water to air.

742 Table B.5 shows the Carnot efficiencies η_{Carnot} used for the COP calculation.
743 As described in [35], COP_h is limited by 7 and COP_c by 6.

744 All technical parameters of the ice storage are listed in Table B.6. Eco-
745 nomic parameters are shown in Table B.7.

746 For the use case presented in Section 3.1, the annualized costs of the
747 thermal network are $C_{\text{netw}} = 28.3 \text{ kEUR/a}$. The total heat transmittance of
748 the network is $(kA)_{\text{tot}} = 3.66 \text{ kW/K}$. Hydraulic pumps with a total electric
749 capacity of 9.32 kW and a total annual electricity demand of 10.44 MWh are
750 installed. As explained in Section 4.1, 63 demand scenarios are generated
751 by defining subsets of buildings clusters. In the original model formulation,
752 annualized costs for the network infrastructure (pipe costs and earthworks)
753 as well as pumping work are considered, which both are calculated prior to
754 the optimization. For evaluating the demand scenarios, a network topology
755 is not designed for each demand scenario a priori. Instead, the pump work,
756 network costs and the heat loss coefficient of the network (kA_{tot}) , which
757 were calculated for the use case with 17 buildings, are scaled linearly with
758 the total heating and cooling demands of the respective demand scenario.
759 The resulting error is small since the pump work, heat losses and network
760 costs are small compared to the other energy flows and cost proportions [35].
761 For the CO₂ emission calculation, a CO₂ factor of 516 g/kWh is assumed for
762 imported power from the electricity grid.

763 **Appendix C. Optimal energy system design**

764 The optimal energy system design for the use case in Section 3 is listed
765 in Table C.8 (energy hub) and Table C.9 (building energy systems). The

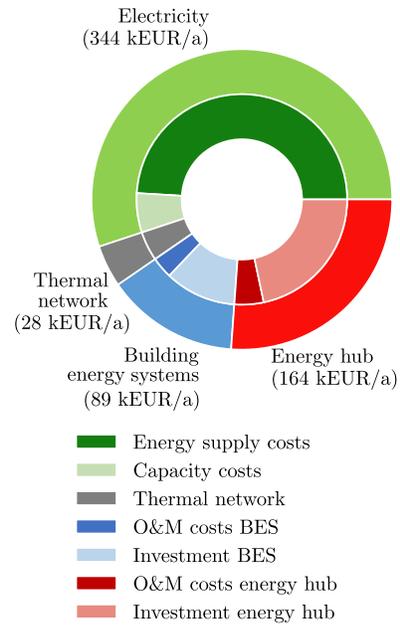


Figure C.17: Proportions of total annualized costs.

766 proportions of the total annualized costs are depicted in Fig. C.17.

Table A.4: Building data and demands.

Buildings	$\dot{Q}_{h,dem}^{tot}$ [MWh]	$\dot{Q}_{c,dem}^{tot}$ [MWh]	Function —	R —	Net floor area m^2
1	1352	490	Office/Lab	0.47	3166
2	1209	84	Canteen	0.87	4118
3	302	3426	Data center	-0.84	4235
4	685	3920	Data center	-0.70	6923
5	455	119	Laboratory	0.59	954
6	640	220	Office/Lab	0.49	2559
7	138	38	Office/Lab	0.57	665
8	91	15	Laboratory	0.72	1106
9	497	1063	Office/Lab	-0.36	7996
10	33	71	Office	-0.37	888
11	35	129	Office	-0.58	2968
12	350	168	Laboratory	0.35	1188
13	239	0	Office/Lab	1	4854
14	107	109	Laboratory	-0.01	240
15	5	131	Laboratory	-0.93	401
16	21	59	Office	-0.47	1210
17	201	0	Office	1	1371

Table B.5: Carnot efficiencies for COP calculation.

	ASHP	ISHP
$\eta_{Carnot} [-]$	0.4	0.5

Table B.6: Technical parameters of the ice thermal energy storage.

$\eta^{\text{ch}} [-]$	$\eta^{\text{dch}} [-]$	$\dot{Q}^{\text{ch,max}} [\text{MW}]$	$\phi_{\text{loss}} [-]$
0.95	0.95	0.3	0.001
$T [^{\circ}\text{C}]$	$s^{\text{min}} [-]$	$s^{\text{max}} [-]$	$S^{\text{cap,max}} [\text{MW}]$
0	0	1	10

Table B.7: Economic parameters.

	ASHP	ITES	ISHP
Specific investment i $[\frac{\text{kEUR}}{\text{MW}}]$	350	15	250
Service life t_L [a]	20	20	20
Capital rec. factor a_{inv} [%]	8.02	8.02	8.02
Share for o&m f_{om} [%]	2.5	2	2.5

Table C.8: Installed capacity and operation of components in energy hub.

Technology	Capacity	Generation	Full load hours $[\frac{\text{h}}{\text{a}}]$
Air source heat pump	1.95 MW _{th}	6042 MWh _{th}	3099
Ice thermal energy storage	4.95 MWh	—	—
Ice storage heat pump	0.04 MW _{th}	60 MWh _{th}	1506
Photovoltaics	1.02 MW _{peak}	1122 MWh _{el}	1100
Battery	—	—	—

Table C.9: Installed capacity and operation of components in building energy systems.

Technology	Capacity	Generation	Full load hours $\left[\frac{\text{h}}{\text{a}}\right]$
Heat pump	1.63 MW _{th}	6400 MWh _{th}	3926
Electric boiler	0.78 MW _{th}	12 MWh _{th}	15
Comp. chiller	—	—	—
Direct cooling	2.88 MW _{th}	10042 MWh _{th}	3487
Thermal energy storage	2.44 MWh	—	—

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